

Università degli Studi di Padova

# Development of a Fine Steering Tip/Tilt Mechanism for Space Applications

Armando Grossi - 36th Cycle

Supervisor: Prof. Ugo Galvanetto

Industrial Supervisor: Eng. Emanuele Piersanti

PhD Course in Science, Technologies and Measurements for Space Admission to second year - 08/09/2021



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INDUSTRIAL DOCTORATE







OFFICINA STELLARE

an innovative SME active in the design and production of telescopes, optomechanical and aerospace instrumentation for Ground and Space based applications











#### **SPACE TELESCOPES**

- Earth observation
- Outer space observation

#### PROBLEM

Line-of-sight and image performances affected by:

- Thermal gradients;
- Misalignment due to launch vibrations;
- Platform jitter (due to reaction wheels);
- Fuel slosh;
- Ground errors (manufacturing, integration, ...)

A SOLUTION active mechanism to adjust the position/orientation of optical elements

#### NECESSITY TO CORRECT OPTICAL COMPONENTS POSITION







Design and Realization of a fine steering mechanism equipped with piezoelectric actuators

**Space qualification** of the fine steering mechanism

Acquisition of know-how in the **piezoelectric actuators** field

Acquisition of experience in the **active optics** field





## An Overview of the Mechanism

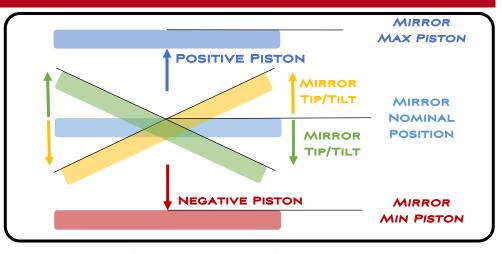




 Movement generated by piezoelectric actuators:

#### o PROs

- High resolutions
- No stick-slip
- No lubrifications
- High vacuum operations
- Low power consumption
- Low heat dissipation
- CONs
  - Limited stroke
  - High Voltage





#### [from Physik Instrumente]







- First year activities completed:
  - ✓ Requirement definition
  - ✓ Bibliographic review
  - ✓ System configuration trade-off
  - ✓ Design and analysis of the amplification mechanism
  - Preliminary design and analysis of the system (amplified piezo actuators + optical payload + IF platform)

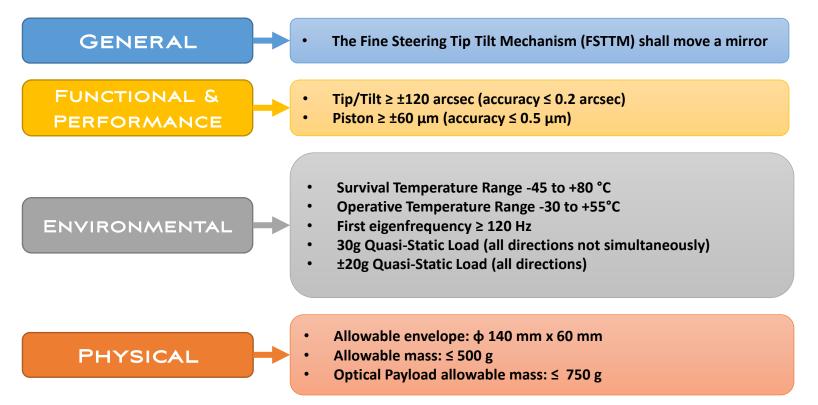




## **Requirements Definition**



• A definition of the Fine Steering Tip/Tilt Mechanism requirements has been performed



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## **Bibliographic Review**



- A bibliographic review have been conducted on:
  - Piezoelectric actuators
  - 3 DoF mechanisms with piezoelectric actuators

Particular attention on:

- those mechanisms performing one translation, and two rotations
- those mechanisms used (or that will be used) in space missions



[SODISM Pointing Mechanism, from Meftah, et. Al., 2011]





[**ATLID BSA**, from Claeyssen, et. Al., 2018]



[**Point Ahead Mechanism** (PAM), from Guignabert, et. Al., 2020]





## System configuration trade-off

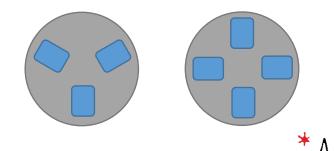


- A system configuration trade-off, to define **type** and **number** of piezo actuators has been done.
- TYPE:
  - Multi-layer piezo (or piezo stack) actuators selected → if equipped with an amplification mechanism, they guarantee both high free displacements and reasonable free force values.



#### • NUMBER:

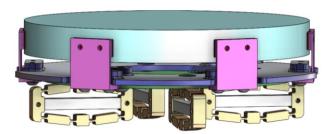
- From literature review, 3-DoF mechanisms, able to perform 2 rotations and 1 translations, can be equipped with 3 or 4 piezo actuators
- 4 actuator configuration selected
  - o Stiffer system
  - o Control system easier
  - Lower required stroke for tip/tilt



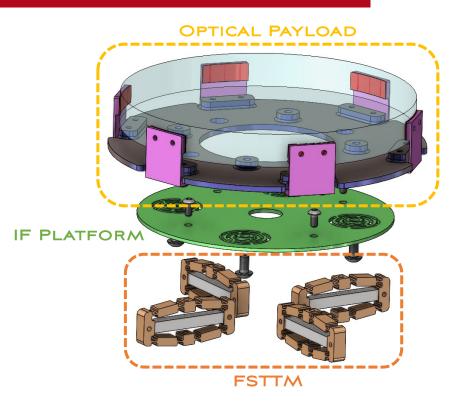


### Preliminary System Design – an overview





- Three sub-systems can be identified:
  - Fine Steering Tip/Tilt Mechanism (FSTTM), consisting of 4 amplified piezo stack actuators
  - Optical Payload, consisting of a mirror, and its supporting cell
  - IF platform, connecting the FSTTM to the optical payload



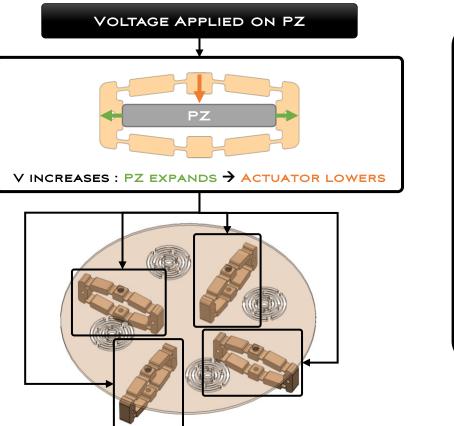


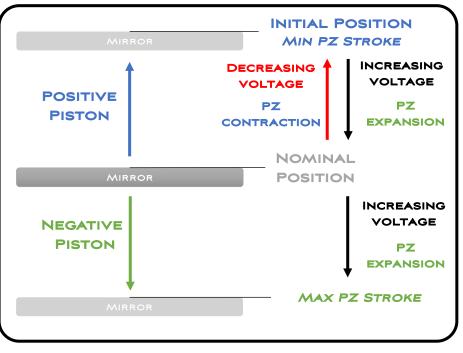


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### **Preliminary System Design – Piston Principle**







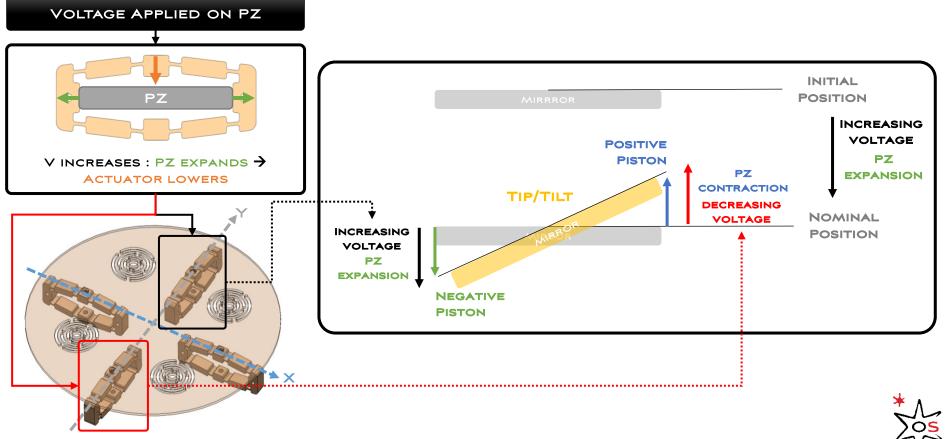




### **Preliminary System Design – Tip/Tilt Principle**



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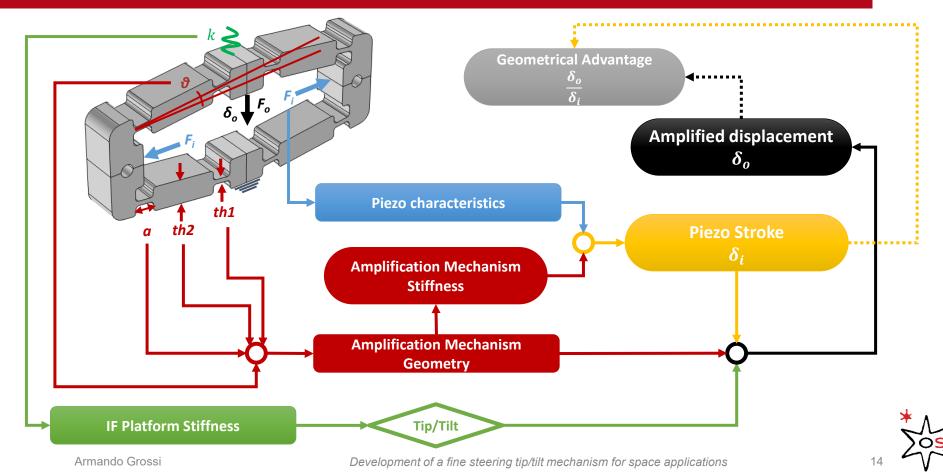


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### **Amplification Mechanism – Parametric Design**

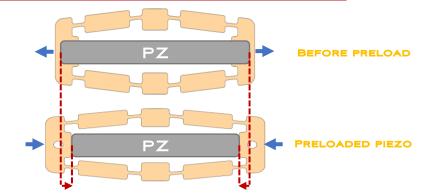




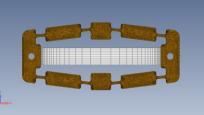




- Pizoelectric actuator has to be preloaded.
- A preload ≤ 15 MPa is recommended by piezo manufacturer.
- Preload realized through the Amplification
  Mechanism → AM axial length < Piezo length</li>



LOAD CONDITION	Average pressure [MPA]	PRESSURE VARIATION [MPA]
20°C (no gravity)	9.12	/
-45°C (no gravity)	14.4	+5.31
-30°C (no gravity)	13.3	+4.16
+ 55°C (no gravity)	6.04	-3.07
+ 85°C (no gravity)	4	-5.12
gravity	8.65	-0.47



/:1		1,67E+7
2		1,6E+7
ā ī		1,53E+7
		1,47E+7
		1,4E+7
		1,33E+7
		1,26E+7
		1,19E+7
		1,12E+7
		1,05E+7
		9,85E+6
		9,16E+6
		8,48E+6
4		7,79E+6
X Y		7,1E+6
Dutput Set: STATIC - 1G, Simcenter NASTRAN Case 1		6,42E+6
Idal Contour: Contact Pressure		5,73E+6

- A preload compression of about 9 MPa has been selected.
- FE analyses performed to analyze the preload variation due to gravity and thermal loads.



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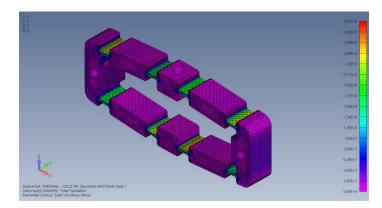


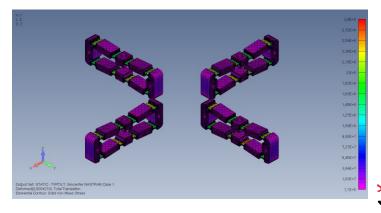
### **Amplification Mechanism – Fatigue Analysis**



- A fatigue analysis on the amplification mechanism has been performed.
- Its endurance (or fatigue) limit has been estimated, considering degradation due to surface condition, reliability and other miscellaneuos effects.
- Assuming a worst case scenario, the amplification mechanism can survive to an infinite number of cycles.
- Stress values from system FEA have been used.

LOAD CONDITION	MAX TENSILE STRESS ON AMPLIFICATION MECHANISM [MPA]
Only Preload (0 V), @ T <sub>amb</sub>	219
Min Piston (-60 μm), @T <sub>amb</sub> – preload included	318
Max Piston (+60 μm), @T <sub>amb</sub> – preload included	231
May Tin (130 areas) @ T neological included	290 (max value)
Max Tip (120 arcsec), @ T <sub>amb</sub> – preload included	259 (min value)
Hot Survival Temperature (0 V) – included preload	104
Cold Survival Temperature (0 V) – included preload	334





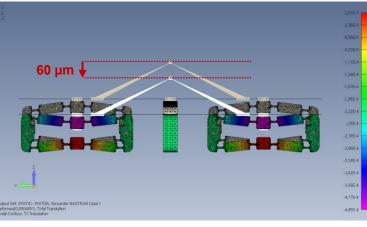


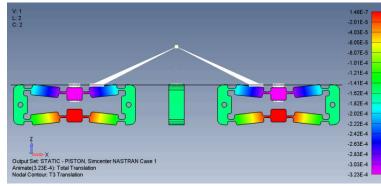
## **Preliminary System Design - FEA**



- The preliminary system designed, has been analysed with several FE analyses.
- FE model consisting in:
  - Amplification mechanisms without piezo, but replaced by their forces;
  - IF platform, connected to the AMs through rigid elements;
  - Optical payload modelled as a point mass, and connected to the IF platform through a rigid element.
- Performed analyses:
  - A maximum tip/tilt case (simulating a rotation of 120 arcsec)
  - Maximum and minimum piston case (simulating a translation of ±60 μm)
  - Modal analysis
  - Stress analysis

#### Piston Case: ±60 µm





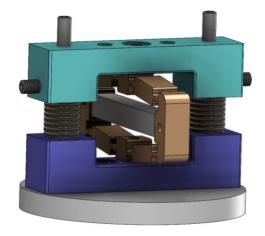


## Next Steps in 2nd Year



## □ Realizazion of a **breadboard**

- consisting in:
  - Amplification Mechanism
  - Piezoelectric Actuator
  - Voltage amplifier and controller
  - GSE to simulate payload stiffness
- to verify results obtained by FEA



### □ From test results, a **critical design** of the system will be performed.

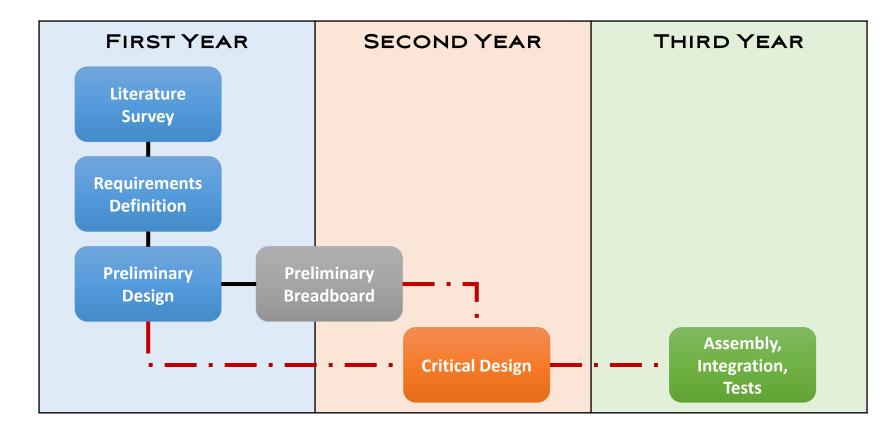




### Work Activity – Main Steps



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# **Thanks for the attention**



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